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Nobuhiro Suzuki
University of Rhode Island

Tetsu Hara
University of Rhode Island, thara@uri.edu

See next page for additional authors

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Impact of Breaking Wave Form Drag on Near-Surface Turbulence and Drag Coefficient over Young Seas at High Winds

NOBUHIRO SUZUKI AND TETSU HARA
Graduate School of Oceanography, University of Rhode Island, Narragansett, Rhode Island

PETER P. SULLIVAN
National Center for Atmospheric Research, Boulder, Colorado

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ABSTRACT

The effects of breaking waves on near-surface wind turbulence and drag coefficient are investigated using large-eddy simulation. The impact of intermittent and transient wave breaking events (over a range of scales) is modeled as localized form drag, which generates airflow separation bubbles downstream. The simulations are performed for very young sea conditions under high winds, comparable to previous laboratory experiments in hurricane-strength winds. The results for the drag coefficient in high winds range between about 0.002 and 0.003. In such conditions more than 90% of the total air–sea momentum flux is due to the form drag of breakers; that is, the contributions of the nonbreaking wave form drag and the surface viscous stress are small. Detailed analysis shows that the breaker form drag impedes the shear production of the turbulent kinetic energy (TKE) near the surface and, instead, produces a large amount of small-scale wake turbulence by transferring energy from large-scale motions (such as mean wind and gusts). This process shortcircuits the inertial energy cascade and results in large TKE dissipation (integrated over the surface layer) normalized by friction velocity cubed. Consequently, the large production of wake turbulence by breakers in high winds results in the small drag coefficient obtained in this study. The results also suggest that common parameterizations for the mean wind profile and the TKE dissipation inside the wave boundary layer, used in previous Reynolds-averaged Navier–Stokes models, may not be valid.

1. Introduction

In this paper, we investigate the turbulence in the atmospheric surface layer that develops over a field of breaking surface waves in hurricane-strength winds (30–70 m s\(^{-1}\)). Such turbulence is important as it affects air–sea exchanges of momentum and heat as well as suspension and dispersion of sea-spray droplets and other passive tracers. These surface-layer processes are critical factors affecting larger-scale phenomena such as tropical cyclones. Despite their importance, the surface-layer processes at high winds remain poorly understood due to the extreme air–sea conditions involved.

An outstanding question is how the drag coefficient \(C_{D_{10}} = (U_* / U_{10})^2\) over the ocean depends on the wind speed at high wind speeds. Here, \(U_*\) is the friction velocity, and \(U_{10}\) is the mean wind speed at 10-m height. In low to moderate winds, the drag coefficient is known to increase with the wind speed (e.g., Edson et al. 2007). However, in hurricane-strength winds, field observations suggest that the drag coefficient may saturate (i.e., stop increasing) or even decrease with the wind speed (Powell et al. 2003; French et al. 2007; Bell et al. 2012) and is much less than the extrapolations of the bulk relationships derived from the low to moderate wind observations. Similar dependence of the drag coefficient on the wind speed has been observed in a fixed-fetch wind–wave tank experiment (Donelan et al. 2004) as well.

The cause of the drag coefficient reduction remains unclear. Possible causes considered in the literature include sea foam, sea spray, and breaking waves. Sea foam (or foam spray) may affect the drag coefficient via altering the velocity boundary conditions for the surface layer (Powell et al. 2003; Soloviev and Lukas 2010; Holthuijsen et al. 2012). Sea spray is a potential cause...
because 1) its mass and its exchange of heat with surrounding air influence the stratification of the surface layer (e.g., Bianco et al. 2011; Bao et al. 2011; Kudryavtsev and Makin 2011), and 2) suspension of spray droplets results in turbulent kinetic energy (TKE) loss and effectively enhances the TKE dissipation rate (Makin 2005; Barenblatt et al. 2005). Both stratification and TKE dissipation rate may modify the turbulence affecting the drag coefficient. Lastly, breaking waves may play a role in the drag coefficient reduction because they affect the atmospheric wave boundary layer (WBL) dynamics. Here, the atmospheric WBL refers to the lower part of the surface layer where airflow is directly affected by waves. Previous theoretical studies of the WBL in high winds (Kudryavtsev and Makin 2007; Kukulka et al. 2007; Kukulka and Harai 2008b; Mueller and Veron 2009) investigated the “sheltering effect” due to airflow separation over breaking waves. Here, the sheltering effect refers to reduction of the viscous surface stress and the form drag of small roughness elements inside an airflow separation bubble formed by a larger breaking wave.

In addition to the above mechanisms, the drag coefficient may be reduced by the vigorous production of wake turbulence over breaking waves and the resultant shortcut of the energy cascade. In high winds, breakers may cause vigorous wake eddies (such as separation bubbles) whose sizes roughly scale with the breaker heights (e.g., Reul et al. 2008). Such wake production transfers energy from large-scale motions (viz., the mean wind and large-scale eddies) to small-scale turbulence near the viscous dissipation scale (i.e., it shortcuts the inertial energy cascade) and results in enhanced TKE dissipation (e.g., Shaw and Schumann 1992; Finnigan 2000). Although the importance of wake-generated turbulence has been long recognized in studies of canopy-layer flows (e.g., Raupach and Shaw 1982), it has been overlooked in the previous theoretical studies of the WBL in high winds.

Another weakness of the previous theoretical WBL studies is that they are based on a Reynolds-averaged Navier–Stokes (RANS) modeling framework and use parameterizations that are originally developed for turbulence parameterizations in the WBL RANS models valid. These questions are answered by using large-eddy simulation (LES) that explicitly simulates intermittent and transient form drag and wake turbulence due to individual breakers. The advantage of such LES over RANS approaches is that it does not heavily rely on turbulence parameterizations other than the subgrid-scale parameterization. In contrast, RANS models have to parameterize the effects of wake turbulence. As our focus is on the breaker form drag and wakes, we will not consider sea foam, sea spray, heat flux, and stratification.

2. Methods

a. LES model of the WBL with breaker effects

Our LES employs an approach successfully used in large-eddy simulations of canopy-layer flows (e.g., Shaw and Schumann 1992) and upper-ocean boundary layer flows (e.g., Sullivan et al. 2007). In such an approach the actual geometry and motion of roughness elements are not resolved, but their impact is modeled by applying local and instantaneous forces that would result from the roughness elements. The force applied in the computational domain interior represents the form drag over intermittent breakers or, more precisely, the momentum exchange between the breakers and their surrounding air via the pressure force induced by the breakers. The LES equations are otherwise standard.

Namely, the governing equations for filtered (or resolved) motions and subgrid-scale (SGS) kinetic energy are (Deardorff 1980; Moeng 1984; Sullivan et al. 1994, 2007)

$$\frac{\partial \overline{u_i}}{\partial t} = -\frac{\partial \overline{u_i}}{\partial x_j} \frac{\partial \overline{p}}{\partial x_j} - \frac{\partial R_{ij}}{\partial x_j} - \frac{\partial \overline{P}}{\partial x_j} + \sum \overline{A_i}^m, \quad (1)$$

$$\frac{\partial \overline{e}}{\partial t} = -\frac{\partial \overline{e}}{\partial x_j} - R_{ij} \frac{\partial \overline{u_i}}{\partial x_j} + T^{SGS} - \epsilon + \sum W^m. \quad (3)$$
Here, filtered variables are denoted by an overbar; \( x_1, x_2, x_3 \) (or equivalently \( x, y, z \)) are the streamwise, spanwise, and vertical coordinates, respectively; \( (u_1, u_2, u_3) = (u, v, w) \) are the velocity components; \( p \) is the pressure divided by the uniform density; \( \bar{P} = \bar{P}(x_1) \) is an external large-scale forcing used to drive the flow, and \( -\partial \bar{P} / \partial x_1 \) is constant in time, uniform in space, and positive; \( R_i = \bar{\pi}_i - \bar{\pi}_i \) is the SGS stress; \( e = (\bar{\pi}_i - \bar{\pi}_i) / 2 \) is the SGS kinetic energy; \( T^{\text{SGS}} \) is the SGS transport; \( \epsilon \) is the viscous dissipation; \( \overline{\mathcal{A}}_i^m \) and \( W^m \) are the momentum input to the resolved motion and the work done to the SGS turbulence in a local discrete breaking wave event \( m \), respectively. We adopt a flat bottom idealization; that is, we employ a surface-fitted coordinate (Fig. 1a), but the equations are approximated with the Cartesian forms.

Note that a breaker-induced flow separation shown in Fig. 1b appears as Fig. 1c in the surface-fitted coordinate system of our LES. In Eqs. (1) and (3), the regular SGS terms (viz., \( R_i \), \( T^{\text{SGS}} \), and \( \epsilon \)) and the breaker effect terms (viz., \( \overline{\mathcal{A}}_i^m \) and \( W^m \)) require modeling. The regular SGS terms are modeled using a conventional TKE-closure SGS parameterization described by Moeng (1984). Some LES runs are repeated using another TKE-closure SGS parameterization described by Sullivan et al. (1994) in order to investigate the sensitivity of our results to different SGS parameterizations. In both SGS parameterizations, the SGS stress is modeled with eddy viscosity \( \nu_T \) diagnosed based on \( e \); \( T^{\text{SGS}} \) is modeled as downgradient diffusion of \( e \), namely \( \frac{\partial}{\partial x_j} (2 \nu_T \partial e / \partial x_j) \); \( \epsilon \) is assumed to be proportional to \( e^{3/2} \). Modeling for the breaker effect terms is described next.

The momentum input \( \overline{\mathcal{A}}_i^m \) is specified in such a way that it models localized forcing and wake production occurring in breaking wave event \( m \). When wind blows over and around a breaker, a localized pressure perturbation appears at the air–sea interface and in the interior of the air surrounding the breaker. This pressure perturbation at the air–sea interface causes the form drag acting on the breaker. The net pressure gradient force on the surrounding interior airflow takes energy and momentum away from the mean wind and gusts. The aim of \( \overline{\mathcal{A}}_i^m \) is to apply this breaker forcing in our LES and to induce energy transfer from the mean wind and gusts to the wake turbulence.

For this reason, \( \overline{\mathcal{A}}_i^m \) is defined in the following manner. First, we estimate the form drag acting on a cross section of breaker \( m \) (Fig. 1a) based on a conventional aerodynamic drag formula (e.g., Kukulka et al. 2007):

\[
F_i^m = \rho a C_d^{BR} |\mathbf{u}^T| - \mathbf{c} |(u_i^T) - c_i| \tag{4}
\]

where \( F_i^m \) is the form drag (per unit breaker crest length) acting on the cross section, \( \rho \) is the air density, \( a \) is the amplitude of the breaker, \( C_d^{BR} \) is an empirically determined form drag coefficient of the breaker, \( \mathbf{c} \) is the propagation velocity of the breaker and is assumed to be related to the wavenumber \( k \) and the gravitational acceleration \( g \) by \( c = \sqrt{g/k} \), and \( \mathbf{u}^T \) is a measure of the wind forcing on the breaker cross section. Specifically, \( \mathbf{u}^T \) is set to the instantaneous upstream wind normal to the breaker crest and is parallel to \( \mathbf{c} \) located at \( z = a \) away from the surface. If \( \mathbf{u}^T \) is opposite to or slower than \( \mathbf{c} \), then \( F_i^m \) is set to zero. In this study, we assume that the breaker slope \( ak \) is 0.3 for all breakers (i.e., \( a \) is set equal to 0.3/\( k \) in our simulations). Note that the range of \( ak \) is generally confined between 0.1 and 0.5 (Kukulka et al. 2007). Next, we apply the same drag force (with an opposite sign) to the airflow. The drag force is uniformly distributed inside an empirically determined area \( V^m \) such that \( \overline{\mathcal{A}}_i^m \) inside the area is

\[
\overline{\mathcal{A}}_i^m = -\frac{F_i^m}{\rho V^m}. \tag{5}
\]

Outside the area, \( \overline{\mathcal{A}}_i^m = 0. \)
The form drag coefficient $C_{BR}^d$ in Eq. (4) and the area $V^m$ are empirically determined so that the wakes produced in LES are comparable with the breaker-induced wakes observed in the laboratory experiment by Reul et al. (2008). Reul et al. find the following wake characteristics.

1) The wakes are unstable and transient.
2) Often there are multiple recirculation vortices in a flow separation bubble.
3) Often an upward burst of air motion is induced near the downstream side of a reattachment point.
4) Generally, the degree of flow separation depends on the wind forcing intensity and the type of breakers such as microbreaking, spilling, and plunging breaker.
5) The maximum backflow speed can reach about 20% to 30% of the mean wind speed at the crest height (i.e., approximately 20% of the free stream wind speed in their tank).
6) The height of the separation bubble is about the height of the breaker amplitude to the breaker height, and the downwind extent of the separation bubble is about 30% to 100% of the wavelength for breakers whose $ak$ is about 0.3.

After exploring different forcing configurations, we have found that the intensity of the recirculation vortices in our LES is mainly controlled by $C_{BR}^d$, that the size of the separation bubbles is mainly controlled by $V^m$, and that the above wake characteristics are well reproduced when the form drag coefficient $C_{BR}^d$ is in the range $0.6 < C_{BR}^d < 3.0$, and the area $V^m$ is a rectangle as shown in Fig. 1d, where $\lambda = 2\pi/k$ is the wavelength of the breaker. In addition, at the locations where $(c_i - \bar{v}_i)\bar{A}_m$ is negative, $\bar{A}_m$ is reset to 0. This is done to avoid an unphysical (negative) value for the SGS work input $W_m$, as explained later. In reality, as the intensity and geometry of a breaker are variable and transient, $C_{BR}^d$ and $V^m$ are likely variable and transient as well. However, when we use a static and constant $C_{BR}^d$ and a static $V^m$ whose scale is proportional to the breaking wavelength, the simulated wake turbulence is sufficiently unsteady and variable to resemble the foregoing wake characteristics observed by Reul et al. (2008). Introducing variability and unsteadiness in $C_{BR}^d$ or $V^m$ does not change our overall results. Figure 2 shows examples of resolved-scale wakes produced in our LES. In the following, we set the baseline (default) value of $C_{BR}^d$ to be 1.0 and use different values of $C_{BR}^d$ (viz., 0.6 and 3.0) only when we investigate the sensitivity of our $C_{D10}$ results to $C_{BR}^d$.

Finally, modeling for $W_m$ in Eq. (3) is done in such a way that the overall energy transfer occurring in a breaking wave event is conserved; that is, the rate of energy loss in the resolved wind is balanced by the rate of energy gain in the SGS turbulence and the rate of energy
transfer to the breaker. According to Eq. (1), the rate of total work done by breaker forcing on resolved winds in breaking wave event $m$ is given by $\int \mathbf{u} \cdot \mathbf{A}^m dx dy dz$ where the integral is taken over the region forced by the breaker. On the other hand, the rate of energy transfer to the breaker may be estimated by the breaker propagation velocity times the form drag, namely $-c_i \int \mathbf{A}^m_i dx dy dz$ (e.g., Kukulka et al. 2007). Then, the conservation of energy may be written as $\int (W^m - c_i \mathbf{A}^m_i + \mathbf{u} \cdot \mathbf{A}^m) dx dy dz = 0$. To satisfy the energy conservation, we simply model $W^m$ as $W^m = (c_i - \mathbf{u}) \mathbf{A}^m_i$ in this study. Note that the SGS wake production $W^m$ represents the energy transfer to SGS motions from resolved motions. It is unphysical if this term is negative (i.e., if SGS motions convert into large-scale motions by breakers). Therefore, when $W^m = (c_i - \mathbf{u}) \mathbf{A}^m_i$ is locally negative, we set $\mathbf{A}^m_i = 0$ there.

### b. Wave age and average air–sea momentum flux considered

In this study, we consider wave conditions that appear in a wind–wave tank at high winds. The reasons are threefold. First, in such conditions waves are narrow banded; that is, the entire range of breaking waves can be explicitly considered using computational domains of reasonable size and resolution. Second, we will find that the total momentum flux (wind stress) is mostly supported by the breaking wave form drag, and the contributions from the surface viscous stress and the nonbreaking wave form drag are practically negligible. Third, in the laboratory conditions the drag coefficient $C_{D10}$ is accurately known and can be compared with the LES results. (In contrast, the drag coefficient in the open ocean is still poorly constrained.) To the best of our knowledge, the wind–wave tank experiment by Donelan et al. (2004) is the most comprehensive experiment at hurricane-strength winds. Their results show the average air–sea momentum flux and the corresponding $U_{10}$ or equivalently $C_{D10}$. In addition, they report the peak wave frequency at the highest wind speed used in their experiment. The peak wave frequency and average air–sea momentum flux can be used to compute the wave age $c_p/U_a$, where $c_p$ is the phase speed at the peak wave frequency. The estimation is made using an empirical relationship between wave age (or inverse dimensionless peak wave frequency) and dimensionless fetch: namely, $c_p/U_a \approx (X_f/gU_a^3)^\alpha$ in which $X_f$ is the fetch and $\alpha$ is a constant ranging $0.23 < \alpha < 0.33$ (Babanin and Soloviev 1998).

### c. Field of breakers

During our LES runs, discrete breaking wave events over a range of wavenumbers are generated intermittently in time, randomly in space, and independently from the airflow. Once generated, each breaking wave event lasts for one wave period $2\pi/\sqrt{gk}$, and its position moves at its breaker propagation velocity $c$. The spanwise dimension of each breaking wave event is set to its wavelength. These parameter choices follow Sullivan et al. (2007) and Suzuki et al. (2011), and our results are relatively insensitive to the particular choices made here. A random number of breaking wave events at each wavenumber are initiated at each time step in such a way that the resultant breaker field satisfies a specified breaking wave distribution function $\Lambda(k, \sigma)$ on a long time average over the entire bottom boundary. Here, $\sigma$ is the breaker propagation direction, and $\Lambda(k, \sigma)dkd\sigma$ represents the average length of breaking crests per unit horizontal area of the sea surface for waves with wavenumbers between $k - dk/2$ and $k + dk/2$ and propagation directions between $\sigma - d\sigma/2$ and $\sigma + d\sigma/2$ (e.g., Phillips 1985; Kleiss and Melville 2011).

Unfortunately, there are scant experimental observations of $\Lambda$ in hurricane-strength winds. Thus, we specify $\Lambda$ based on the Reynolds-averaged Navier–Stokes wave boundary layer model of Kukulka and Hara (2008b). Their RANS model is based on the conservation of wave energy as well as the conservation of airflow momentum and energy, and it predicts $\Lambda$ for fully developed airflow turbulence over very young to mature seas. The predicted $\Lambda$ is consistent with existing observations in open ocean conditions at low to moderate winds where the wave age is 10 or larger. In higher wind speeds and younger sea states, the model results have not been validated as direct observations of $\Lambda$ are not readily available. Therefore, we consider a wide range of uncertainty in $\Lambda$.

According to their RANS model, the directional spreading of $\Lambda$ becomes narrower for younger seas, and it becomes unidirectional in the asymptotic limit of very young sea states. Thus, in the following we assume unidirectionality; that is, we assume that all breaking waves propagate in the mean wind direction. (We tested different directional spreading cases and found that the results are relatively insensitive to this choice.)

Their RANS results strongly depend on several key parameters (viz., the breaker form drag coefficient, the wave energy dissipation rate due to breaking, the breaker...
wave height, and the sheltering coefficient). Since these parameters are not well constrained, the magnitude and shape of $\Lambda(k)$ are also not well constrained. For example, $\Lambda(k)$ may monotonically increase or decrease with $k$. We therefore test several different breaking wave distribution functions. Figure 3 shows the $\Lambda(k)$ used in our simulations at wave age 0.5. Here, the baseline case (BAS) is determined such that 1) the $\Lambda(k)$ value is between our estimates of the upper and lower bounds of $\Lambda(k)$ (see the appendix). The box shows the estimated range of $\Lambda(k)$ in the laboratory measurement of dominant microbreakers by Jessup and Phadnis (2005).

![Figure 3. Breaking distribution $\Lambda(k)$ used in our LES at wave age 0.5: baseline (BAS), sparse (SPA), dense (DEN), level (LEV), and short-breaker dominating (SHO) cases. The horizontal dot lines are the estimated upper and lower bounds of $\Lambda$ for the dominant waves, which we believe is qualitatively consistent with laboratory observations (Jessup and Phadnis 2005).](image)

**d. Numerical method**

Time integration uses an explicit, third-order, three-substep Runge-Kutta scheme. A fixed time step is used based on a fixed Courant–Friedrichs–Lewy condition ($\Delta t u/\alpha_o = 0.02$ or 0.015 depending on the simulated cases). Horizontal differentiation uses the pseudospectral method. Vertical differentiation uses the second-order centered finite difference method on a vertically staggered grid. The variables $\bar{w}$, $e$, and $W^m$ are stored at the same grid levels (hereafter, $w$-nodes), and $\bar{w}$, $\bar{e}$, $\bar{p}$, $\Lambda_1^m$, and $\Lambda_2^m$ are stored at the grid levels (hereafter, $w$-nodes) located midway between the $w$-nodes. The $w$-nodes hold the bottom and top boundaries. The bottom boundary is at $z = 0$. The grid is horizontally uniform and vertically nonuniform. We locate the fifth $w$ node at $z = a_o$ where $a_o$ is the amplitude of the tallest breaker, and set the distances of the lowest six $w$ spacing to be $\Delta z/a_o = 2/9$. Above this, each $w$ spacing $\Delta z/a_o$ is 1.03 times larger than the spacing one-node below.

The horizontal boundaries are periodic. The top boundary is frictionless and nonpermeable. The bottom boundary is nonpermeable. For the bottom SGS stress, we tested several different parameterizations including a conventional one (Moeng 1984) and find that our results are relatively insensitive to a modest change in bottom SGS stress parameterizations. This is because breaker forcing is responsible for almost the entire air-sea momentum flux, and the mean wind near the water surface is small in all simulations presented in this paper.

The horizontal domain size $L_x \times L_y$ is $L_x/a_o = L_y/a_o = 83.78$; $L_z$ is four times the wavelength of the largest breaking wave considered. The domain height $L_z$ is $L_z/a_o = 56.22$. The grid has 128 x 128 nodes horizontally and 96 nodes vertically.

The initial condition is a small and uniform streamwise wind everywhere. All results are obtained after the flow is converged to a statistically steady (i.e., fully developed) state. Note that, in the current LES, the breaker field ($\Lambda$) is kept constant in time and space (i.e., the wave growth in time or space is ignored). In reality, the wave field evolves in time or space at real young sea conditions; as a result, the airflow turbulence in such conditions may not be horizontally homogeneous nor steady. However, in this study, we assume that airflow at young sea conditions may be approximated with the horizontal periodicity and fully developed state of airflow turbulence.

Some quantities are averaged for the following analysis. The averaging is done over a horizontal plane and over a long time (i.e., $71$ large-eddy turnover time $tU_w/L_z$).
3. Results of low-order moments

a. Mean wind profile and drag coefficient

In the following, angle brackets denote a horizontal average, and a single prime denotes the deviation from it; for example, $u = \langle u \rangle + \prime u$. First, let us investigate the mean wind profiles. Figure 5 shows the normalized mean wind shear $\Phi_m = (z/kU_w) d\langle u \rangle / dz$ at wave age $c_p/U_w = 0.5$. In the figure, the distance from the water surface is normalized with the amplitude of the tallest breaker $a_o$. The results at $c_p/U_w = 0.4$ are not shown since they are essentially identical to the ones shown. The breaker conditions tested are five cases of different $\Lambda$ (BAS, DEN, SPA, LEV, SHO; see Fig. 3) with the default breaker
form drag coefficient \((C_{d}^{BR})\) and two cases of different \((C_{d}^{BR})\) (=0.6 and 3.0) with the baseline \(\Lambda\). In addition, one run (BAS \(\Lambda\) with \((C_{d}^{BR}) = 1\)) is repeated using a different SGS model by Sullivan et al. (1994). The results show little dependence on the different breaker conditions and only weak dependence on the choice of SGS models (near \(z/a_o = 1\)). Thus, the impacts of breakers are robust and not significantly affected by the uncertainties in \(\Lambda\) and \((C_{d}^{BR})\) or the different SGS models.

Away from the surface, the wind profiles are logarithmic (i.e., \(\phi_{m} = 1\)) as expected; the profiles are roughly logarithmic above \(2a_o\) and nearly perfectly logarithmic above \(5a_o\) to \(6a_o\). This height of the log-layer bottom is similar to turbulent flows over other types of roughness (e.g., Ikeda and Durbin 2007). In contrast, the wind profiles are not logarithmic (i.e., \(\phi_{m} \neq 1\)) near the surface in the WBL. The solutions show the existence of three characteristic regions in the atmospheric wave boundary layer: 1) the region well inside the WBL where the wind shear is much less than the log-profile shear, 2) the region near the top of the WBL around \(z/a_o = 1\) where the shear is higher than the log-profile shear, and 3) the region around \(2 \leq z/a_o \leq 5\) where the shear is slightly lower than the log-profile shear.

In the first and second regions, the mean wind is not logarithmic because of the breaker-induced wakes. When the flow separates over a breaker, the region of very high shear that is usually attached on the water surface separates from the surface and appears along the edge of the separation bubble (Fig. 6). Hence, the wind profile spatially averaged at the separation bubble height (i.e., the second region) becomes steeper than the logarithmic wind profile. On the other hand, the local wind inside the separation bubble is much lower than the log-profile shear (Fig. 6). Hence, the spatially averaged wind profile well below the separation bubble height (i.e., the first region) is less steep than the log-profile. The same shear patterns of breaker-induced wakes are also shown in the particle image velocimetry (PIV) images by Reul et al. (2008). In addition, a similar trend of the mean wind shear is observed in the direct numerical simulation (DNS) over \(k\)-type roughness (Ikeda and Durbin 2007).

In the third region \((2 \leq z/a_o \leq 5)\), the shear is slightly lower because the breaker forcing is anisotropic (Suzuki et al. 2011). At young sea states, breaking waves appear mostly perpendicular to the mean wind. Because the pressure form drag is normal to the breakers, the breaker forcing is mostly streamwise, and spanwise turbulent winds experience little drag. Such anisotropic drag results in reduced dissipation of the surface-attached log-layer quasi-streamwise vortices. The enhanced quasi-streamwise vortices, then, result in increased vertical mixing and reduced wind shear (and associated reduced TKE shear production).
In summary, the WBL wind profile is not analogous to the wind profile over a flat wall. It is strongly modified due to the breaker-induced flow separation (first and second regions) and, to a much less extent, the directionality of the breakers (third region). Figure 7 shows the mean wind profiles for the three cases having different breaker densities (DEN, BAS, and SPA) with $C_{\text{BR}}^d = 1.0$. Notice that the overall change in breaker density affects the mean wind speed (and the drag coefficient) even if it hardly affects the mean wind shear.

Next, we show the drag coefficient $C_{D10}$ (Fig. 8). It is computed from the mean wind speed in the log layer above the WBL. Overall, $C_{D10}$ falls in the range between 0.002 and 0.003. If the breaker distribution is kept roughly the same as the wind speed increases, then the $C_{D10}$ remains nearly constant at high winds. As the overall breaker density increases (DEN) or decreases (SPA) compared to the baseline case (BAS), the drag coefficient increases or decreases as expected: $C_{D10}$ increases by about 50% when the amount of breaking events increases by about sixfold (from SPA to DEN). If the breaker form drag coefficient $C_{\text{BR}}^d$ increases or decreases, the drag coefficient $C_{D10}$ also increases or decreases, but the impact is smaller. We also find that varying the $k$ dependence of $\Lambda$ (LEV and SHO cases compared to BAS case) has negligible effects on the drag coefficient (not shown). The effect of varying $\Lambda$ at wave age $c_p/\zeta_w = 0.5$ and $\zeta_w = 2.0$ m s$^{-1}$ (open symbols) is almost identical to that at wave age $c_p/\zeta_w = 0.4$ and $\zeta_w = 2.65$ m s$^{-1}$ (solid symbols). In summary, our LES results of $C_{D10}$ are roughly consistent with laboratory observations although the large uncertainties in $\Lambda$ and $C_{\text{BR}}^d$ yield $C_{D10}$ varying between 0.002 and 0.003.

Since the overall results are not overly sensitive to $C_{\text{BR}}^d$ or the $k$ dependence of $\Lambda$, we examine only the three cases, namely DEN, BAS, and SPA, with $C_{\text{BR}}^d = 1.0$ hereafter.
b. Energy budget of the WBL

The energy budget of the WBL provides valuable insight into why $C_{D10}$ saturates in high winds. Let $E_M = \langle \overline{u} \rangle \langle \overline{u} \rangle / 2$ be the kinetic energy of the mean wind and $E_{RT} = \overline{u} \overline{u}' / 2$ be the TKE of the resolved-scale turbulence. According to Eq. (1) and Eq. (3), in a statistically steady state, the energy budgets of the mean flow, the resolved-scale TKE, and the SGS TKE can be expressed as

$$0 = \frac{\partial \langle E_M \rangle}{\partial t} = \frac{-\partial \langle \overline{u} \rangle \langle \overline{u} \overline{u}' + \overline{R}_{13} \rangle}{\partial z} + \langle \overline{R}_{13} \rangle \frac{\partial \langle \overline{u} \rangle}{\partial z} - \frac{P_{\text{MS}}^m}{\langle \overline{w} \rangle} \frac{\partial \langle \overline{w} \rangle}{\partial x} + \sum_i \frac{\partial \langle \overline{A}_i \rangle}{\partial z} - \langle \overline{\epsilon} \rangle,$$

(6)

$$0 = \frac{\partial \langle E_{RT} \rangle}{\partial t} = \frac{-\partial \langle \overline{u} \rangle \langle \overline{u} \overline{w}' + \overline{R}_{13} \rangle}{\partial z} + \langle \overline{R}_{13} \rangle \frac{\partial \langle \overline{u} \rangle}{\partial z} + \sum_i \frac{\partial \langle \overline{A}_i \rangle}{\partial z} - \langle \overline{\epsilon} \rangle,$$

(7)

$$0 = \frac{\partial \langle \epsilon \rangle}{\partial t} = \frac{-\partial \langle \overline{w} \rangle \langle \overline{w} \overline{w}' + \overline{R}_{13} \rangle}{\partial z} + \langle \overline{R}_{13} \rangle \frac{\partial \langle \overline{w} \rangle}{\partial z} + \sum_i \frac{\partial \langle \overline{A}_i \rangle}{\partial z} - \langle \overline{\epsilon} \rangle,$$

(8)

where $P_{\text{MS}}^m = \sum_i W_m$. Here Eq. (6) is obtained by taking the inner product of $\langle \overline{u} \rangle$ and the horizontal average of Eq. (1). Equation (7) is obtained by subtracting Eq. (6) from the horizontal average of the product of $\langle \overline{u} \rangle$ and Eq. (1). [Note that the last term in Eq. (7) can be written as $\langle \sum_i \overline{u} \overline{A}_i \rangle$, but we prefer the form shown because $A_i = 0$ and $A_i' \neq 0$ outside breaking wave events and it allows an easier physical interpretation.] Equation (8) is simply the horizontal average of Eq. (3). In these equations, $P_{\text{MS}}^m$ and $P_{\text{WS}}^m$ are the production of resolved-scale and SGS turbulence due to the mean-wind shear $\partial \langle \overline{u} \rangle / \partial z$, respectively, and $P_{\text{RIS}}^m$ is the production of SGS turbulence due to the resolved turbulent wind shear $\partial \overline{u}_i / \partial x$. The terms $P_{\text{MS}}^m$ and $P_{\text{WS}}^m$ are the rate of work done by breaker forcing on resolved scale turbulence and SGS turbulence, respectively.

The energy budget of the total energy $E = E_M + E_{RT} + \epsilon$ and the total TKE $E_{TKE} = E_{RT} + \epsilon$ can be obtained using Eqs. (6), (7), and (8):

$$0 = \frac{\partial \langle E \rangle}{\partial t} = \frac{-\partial \langle \overline{u} \rangle \langle \overline{u} \overline{w}' + \overline{R}_{13} \rangle}{\partial z} + \langle f_T \rangle - \langle \epsilon \rangle$$

and

$$0 = \frac{\partial \langle E_{TKE} \rangle}{\partial t} = \frac{-\partial \langle \overline{u} \rangle \langle \overline{u} \overline{w}' + \overline{R}_{13} \rangle}{\partial z} + \langle f_T \rangle + (P_{\text{MS}}^R + P_{\text{WS}}^R)$$

$$+ \left( \sum_i \langle \overline{A}_i' \rangle \right) - \langle \overline{P} \rangle \left( \sum_i \langle \overline{A}_i' \rangle \right) - \langle \epsilon \rangle,$$

(9)

(10)

is the TKE flux. The terms on the rhs of Eq. (10) are called the TKE transport, shear production, wake production, and viscous dissipation, respectively. Note that the fourth term on the rhs of Eq. (9) is the rate of work done by the externally imposed background forcing $-\partial \overline{P} / \partial x$, used to drive the flow. This term does not exist for a turbulent Couette flow and the atmospheric surface layer (i.e., a constant stress layer with no Coriolis effect) since there is no background pressure gradient forcing for these flows. In the current LES, this term is negligibly small in and near the WBL. Hence, our results in and near the WBL are still representative of the energy budget of a constant stress layer.

The relationship between the energy budget and the drag coefficient can be obtained by vertically integrating Eq. (9) from the surface to some height $H_L$ inside the log layer and by considering the overall energy budget in this layer. Note that $\langle \overline{u} \rangle$ and $\langle f_T \rangle$ are either zero or very small at the surface. Thus, omitting these terms at the surface as well as the aforementioned small background forcing term, we can express the normalized mean wind speed at $H_L$ as

$$\frac{\langle \overline{u} \rangle (z = H_L)}{U_0} = \frac{\langle f_T \rangle (z = H_L)}{U_0^3} + \int_{z=0}^{H_L} \frac{\langle \epsilon \rangle}{U_0^3} dz + \int_{z=0}^{H_L} \frac{-\sum_i c_i \overline{A}_i'}{U_0^3} dz.$$

(12)

Here the Reynolds shear stress $\langle \overline{u} \overline{w}' + \overline{R}_{13} \rangle$ at $z = H_L$ is approximated to be $-U_0^2$. This substitution is exact for a constant stress layer. The left-hand side represents the downward energy flux (energy input) $\langle \overline{u} \overline{w}' + \overline{R}_{13} \rangle$ at
the layer top, normalized by \( U_3^* \) (Fig. 9). This energy input is balanced by the right-hand side representing energy outputs from the layer (Fig. 9): namely, the upward TKE flux at the layer top, the TKE dissipation integrated over the layer, and the energy transfer to the breakers integrated over the layer (all terms normalized by \( U_3^* \)). Notice that, by definition, the lhs of Eq. (12) equals \( 1/\sqrt{C_{D10}} \) when \( H_L = 10 \text{ m} \); thus,

\[
C_{D10} = \left( \frac{\langle f_T \rangle(z = 10 \text{ m})}{U_3^*} + \int_{z=0}^{10m} \frac{\langle \epsilon \rangle}{U_3^*} \, dz \right)^{-1} \quad \text{and} \quad \int_{z=0}^{10m} \left( -\sum_{m} \frac{\langle X^m \rangle}{U_3^*} \right) \, dz \right)^{-2}.
\] (13)

In summary, Eqs. (12) and (13) show that for a given wind stress the reference wind is higher or the drag coefficient is lower when the surface layer fluxes out or dissipates more energy.

In all cases of this study we find that the integrated energy transfer to the breakers [the third term on the rhs of Eq. (12)] is much less than the integrated TKE dissipation [the second term on the rhs of Eq. (12)]. It is small because the normalized breaker propagation speed \( c/\bar{U}_w \) (i.e., the wave age) of the laboratory-scale short waves are very small. Likewise, the TKE flux at the layer top [the first term on the rhs of Eq. (12)] is much less than the integrated TKE dissipation. Therefore, the TKE dissipation is the dominant factor in determining the drag coefficient of very young seas in hurricane-strength winds. Most importantly, the \( C_{D10} \) observed in our LES and the laboratory experiment implies that the normalized TKE dissipation in the surface layer is large and saturates in high winds.

This large TKE dissipation is closely related to the large production of small-scale wake turbulence in the WBL. Figure 10 shows the TKE budget for three cases (DEN, BAS, SPA) having different breaker densities. The vertical profiles shown are the four terms on the rhs of Eq. (10) normalized by \( U_3^*/\bar{a}_w \): transport term (solid line), shear production (dash-dot line), wake production (dashed line), and dissipation (dotted line). Cases shown are with wave age = 0.5, \( \bar{U}_w = 2 \text{ m s}^{-1} \), \( C_{BR} = 1.0 \), and three different \( \Lambda \) (DEN, BAS, and SPA).
increases significantly and exceeds the shear production in the lower part of the WBL. Because the wake production in high winds is large enough to replace the reduction of the shear production, the net (i.e., the sum of the shear and wake) TKE production stays large and keeps the TKE dissipation large. As a result, $C_{D10}$ remains small.

While the above statement holds true at any breaker density, the TKE budget also shows a notable dependence on the breaker density. In particular, as the breaker density becomes lower (from DEN to SPA), the wake production and the dissipation become larger (Fig. 10) and the drag coefficient becomes smaller (Fig. 8). The TKE transport also shows some dependence.

c. Validity of existing RANS WBL parameterizations

As mentioned in the introduction, previous studies of the WBL based on RANS modeling rely on turbulence parameterizations derived by analogy to flat-wall turbulence. We have already seen, in Figs. 5 and 7, that the wind profile inside the WBL is significantly different from the log profile, which is the assumed wind profile in some RANS models. The log-profile assumption overestimates the wind speed near the top of the WBL and significantly underestimates the wind speed in the lower part of the WBL. Therefore, such an assumption may lead to erroneous estimates of the drag coefficient.

The RANS models by Kukulka et al. (2007) and Kukulka and Hara (2008a,b) assume that the TKE dissipation is simply related to the Reynolds shear stress $(-\langle\overline{u'w'}+\overline{R_{13}}\rangle)$ as

$$
\langle\epsilon\rangle = \frac{(-\langle\overline{u'w'}+\overline{R_{13}}\rangle)^{3/2}}{k\zeta}
$$

at each height where $\kappa = 0.4$ is the von Kármán constant. In Fig. 11, the TKE dissipation parameterized by Eq. (14) is computed using the LES result of $(\overline{u'w'}+\overline{R_{13}})$ and is compared to the TKE dissipation resulted in our LES. In all cases, the RANS dissipation model significantly underestimates the TKE dissipation, particularly in the lower part of the WBL. The vertically integrated TKE dissipation is also underestimated appreciably. Therefore, these RANS models likely overestimate the drag coefficient.

Above the layer where Eq. (14) underestimates $\langle\epsilon\rangle$, there is a layer where Eq. (14) overestimates $\langle\epsilon\rangle$ (Fig. 11b). This is because the RANS parameterization is designed without accounting for the very small, but nonzero, TKE transport (Fig. 10) and the reduction of the wind shear (Fig. 5) in this layer. However, it is clear from Fig. 11a that the overestimation here is not nearly as important as the underestimation below $z/a_0 \approx 1$.

4. Results of turbulence structures and their characteristics

a. Instantaneous turbulence structures

An example of instantaneous streamwise velocity on a horizontal plane is shown at different heights in Fig. 12. Away from the surface, the turbulence shows the typical streak patterns of shear turbulence (Fig. 12c). These streaks are generated by the quasi-streamwise vortices (including the cane and hairpin vortices) in the log layer (Fig. 13). They are the main turbulence structures of the log layer over flat walls (e.g., del Álamo et al.
In contrast, in Figs. 12a,b the turbulence near the surface is strongly modified by the breaker-induced wakes, and the typical streak patterns no longer exist. The wakes can be identified by the low or negative winds in and past the areas where breaker forcing appears. The wakes show strong three dimensionality (along-crest variability) and are very transient. These features are consistent with the PIV observations of breaker-induced wakes (Reul et al. 2008). Among the wakes, there are sporadic regions of very high wind. These gusty regions roughly match the gusty regions at higher elevations unless the flow separation bubbles prevent such gusty motions near the surface. This suggests that a gust in the WBL comes from outside the WBL in the form of a sweep (i.e., a motion with $\overline{u'} > 0$ and $\overline{w'} < 0$) made by the large-scale quasi-streamwise vortices.

The mixing-layer-type turbulence structures, often seen in canopy-layer flows (Finnigan 2000), are not observed in our results (Fig. 13) even though there is a weak inflection of the mean wind profile very close to the surface (Fig. 7b). The absence of such structures with a mean wind shear inflection is also reported in a DNS study of a flow over transverse $k$-type roughness (Ikeda and Durbin 2007).

b. TKE and variances

In the following, we will investigate how the breaker density affects the turbulence characteristics. Figures 14 and 15 show snapshots of some key turbulent quantities for the dense case and the sparse case at wave age 0.5. Included are streamwise turbulent wind $\overline{u'}/\overline{U}_w$, net TKE...
\( ERT \) and \( U^3 \), vertical velocity \( \bar{w} / U_* \), TKE flux \( f_T / U^3_0 \), ejections (i.e., motions with \( \bar{w} < 0 \) and \( \bar{v} > 0 \)) and sweeps (i.e., motions with \( \bar{v} > 0 \) and \( \bar{w} < 0 \)) expressed as \( \bar{v} \bar{w} / U^3_0 \), and net stress \( (\bar{v} \bar{w} + R_{13}) / U^3_0 \).

The TKE behaves quite differently between the dense and sparse cases. In the dense case, Fig. 14 shows a high correlation among the sporadic gusts (red spots in 14a), large TKE (red spots in 14b), downward TKE flux (blue spots in 14d), and sweeps (blue spots in 14e). This shows that the TKE inside the wave boundary layer is mostly due to the sporadic gusts, and this gust TKE is carried down into the WBL from outside by the sweeping motion associated with the large-scale quasi-streamwise vortices. The TKE of the wakes is much less than the gust
TKE because the wakes cover a large part of the WBL and the mean (horizontally averaged) wind speed is close to the wind speed in the wakes. Hence, the deviations $|\bar{u}'|$ in the wakes are small (Fig. 14a), and the TKE is small as well.

In the sparse case, in contrast, the deviations $|\bar{u}'|$ from the mean wind are large inside the wakes because the mean wind is relatively large (Fig. 15a). Hence the wake turbulence carries more TKE than the sweeps (gusts) (Fig. 15b).

The increased dominance of the wake turbulence in the SPA case is also evident in the variances shown in Fig. 16. The breakers in the sparse WBL result in a very large $\left\langle \bar{u}' \bar{u}' \right\rangle$ whereas breakers in the dense WBL make the flow more uniform with a much smaller $\left\langle \bar{u}' \bar{u}' \right\rangle$. The variance of the cross-stream velocity $\left\langle \bar{v}' \bar{v}' \right\rangle$ stays relatively high inside the WBL in all cases because the breaker form drag is anisotropic as explained earlier.

Fig. 15. As in Fig. 14 but with the sparse (SPA) $\Lambda$. 
c. Wake production: Energy conversion due to breaker form drag

As explained in section 2a, the terms representing work done by the breaker form drag satisfy the following conservation equation:

$$\left\langle \frac{\pi' \pi'}{U_*^2} \right\rangle = \left\langle \frac{\pi' \pi'}{U_*^2} \right\rangle + \left\langle \frac{P_W^R}{P_W^{SGS}} \right\rangle = 0.$$ (15)

The first term represents the rate of energy loss in the mean flow energy ($E_M$) by action of the drag. Since $\bar{A}^{m}_i < 0$ everywhere, the first term is always negative. The second term is the energy transfer to the breakers via the work done by the form drag and is always positive (i.e., waves gain energy). The third term $\left\langle \frac{P_W^R}{P_W^{SGS}} \right\rangle = \left\langle \sum_m c_i \bar{A}^m_i \right\rangle$ is the rate of work done on the resolved-scale turbulence by the form drag and can be positive or negative. For example, resolved-scale gusts have $\pi' > 0$, and the breakers do work against them ($\pi' \bar{A}^m_i = 0$). Hence, the gusts lose energy, and that energy is transferred to the breakers and the SGS wake turbulence. In contrast, $\pi$ is negative inside a wake (Figs. 14a and 15a). Thus, the resolved wake turbulence gains energy ($\pi' \bar{A}^m_i = 0$) from the mean flow. The term $\left\langle \frac{P_W^R}{P_W^{SGS}} \right\rangle$ is the average of these processes and is positive when the energy gain in the resolved wake turbulence is more than the energy loss in the resolved gusts, and vice versa.

Lastly, the fourth term of Eq. (15) is the SGS wake production term and is always zero or positive, as discussed in section 2a. In summary, Eq. (15) states that, when large-scale energetic motions (viz., the mean flow and gusts) hit breakers, they lose energy. Part of that lost energy is transferred to the breakers and the rest is converted to resolved-scale and SGS wake turbulence. Because the size of the wake turbulence roughly scales with the breaker height, the wake turbulence induced by short breakers is close to the viscous dissipation scale. This direct conversion of the mean flow energy and the large-scale TKE to the dissipative-scale TKE shortcuts the usual energy cascade and leads to large energy dissipation (Fig. 17). Such an effect of roughness elements has been well recognized in studies of canopy layers (Finnigan 2000). It is a critical mechanism for rough surfaces to dissipate large amounts of energy.

Figure 18 shows the energy conversion, Eq. (15), for the three cases of DEN, BAS, and SPA. There are significant differences in the wake production and the mean wind energy loss near the surface. In the dense breaker case (Fig. 18a), the rate of the mean-flow energy loss decreases near the surface because the mean flow is very small near the surface (Fig. 7). In contrast, when the breakers are sparse, the mean flow very near the surface is about five times larger (Fig. 7). As a result, both longer and shorter breakers are well exposed to high wind and contribute greatly to the conversion from the large-scale motions to the wake turbulence.
d. Shear production and stress

As shown in Fig. 10, the shear production $P_{MS}^{BR} + P_{SGS}^{BR}$ decreases significantly inside the WBL in all cases. This is because the force exerted by breakers impedes both wind shear $\partial(\overline{ww})/\partial z$ (Fig. 5 and 7) and Reynolds shear stress $\langle \overline{ww} + \overline{R_{13}} \rangle$ well inside the WBL. The reduction of the Reynolds shear stress is an inevitable result of the momentum budget obtained by horizontally averaging the momentum equation (1) in a statistically steady state. Above $z/a_o = 1$, the breaker forcing $\langle \sum m \overline{A_{1m}} \rangle$ is zero, and the Reynolds stress profile is determined solely by the constant background mean pressure gradient forcing. Below $z/a_o = 1$, $|\langle \sum m \overline{A_{1m}} \rangle|$ is much larger than $|\partial \overline{P}/\partial x|$. Thus, the Reynolds stress inside the WBL is determined by the breaker forcing. As $\langle \sum m \overline{A_{1m}} \rangle$ is negative, the Reynolds stress is reduced toward the surface. The breaker forcing is often expressed in terms of the breaker stress $\tau_{BR}^{\varepsilon^2}$, where

$$\frac{d(\overline{ww} + \overline{R_{13}})}{dz} = -\frac{\partial \overline{P}}{\partial x} + \langle \sum m \overline{A_{1m}} \rangle$$

Examples of these stresses are shown in Fig. 19. In all cases $\tau_{BR}^{\varepsilon^2}/U_2^2$ reaches nearly −1 near the surface; that is, almost all air–sea momentum flux is supported by the breakers in our LES (i.e., more than 95% for most cases and about 90% for the SPA case).

The mechanisms of the turbulent momentum transfer are different depending on the breaker density. In the
dense breaker case, the downward momentum flux \( u'w' + R_{13} \) (the blue color in Fig. 14f) is not correlated with the wake turbulence (the blue color in Fig. 14a) but is mainly due to sweeps and ejections (the blue color and the red color in Figs. 14e) caused by the quasi-streamwise vortices above the WBL. The sweeps (blue) are more vigorous than the ejections (red). Figure 20 also shows that the sweeps are more vigorous than the ejections, particularly in the upper part of the WBL. Here, when ejections \( (w' > 0) \) are stronger than sweeps \( (w' < 0) \), \( \langle w'^3 \rangle / U_*^3 \) becomes more positive, and vice versa. This is opposite to the state in the logarithmic layer above, where ejections are larger than sweeps.

In the sparse breaker case, inside a breaker-induced separation the spanwise vortex (Fig. 2) causes a large ejection \( (\pi' < 0, w' > 0) \) where the swirling motion goes up (the red color inside the breaker forcing regions in Fig. 15e), yielding a large downward momentum flux (the blue color inside the breaker forcing regions in Fig. 15f). However, near the reattachment point of the same vortex, the vertical velocity changes its sign \( (\pi' < 0, w' < 0) \) and yields upward momentum flux (the red color appearing right behind the form drag regions in Fig. 15f). On average, these negative and positive momentum fluxes in the wakes cancel out. Therefore, the wakes contribute little to the Reynolds shear stress in the sparse case as well. Outside the wakes, the ejections and sweeps are more regular (Fig. 20) compared to the dense breaker case, reflecting less disruption of the regular quasi-streamwise vortex processes (Figs. 15e and 15f).

5. Concluding remarks

Using LES, which resolves individual wakes generated by breaking waves, the impacts of breaker form drag on airflow turbulence and drag coefficient have been studied at young sea states in hurricane-strength winds. Overall, the simulated \( C_{D_{10}} \) falls in the range between 0.002 and 0.003. It remains nearly constant at high winds if the breaker distribution is kept roughly the same as the wind speed increases. The relatively low \( C_{D_{10}} \) results because the normalized TKE dissipation rate integrated over the atmospheric wave boundary layer (WBL) is relatively large in high winds. The main impact of the breaker form drag on the TKE budget is to impede the shear production and, instead, produce small-scale wake turbulence by converting the kinetic energy of the mean wind and large-scale gusts. This shortcut of the usual energy cascade has been known in canopy-layer studies but has been overlooked in previous WBL studies. Because the increased wake production replaces the decreased shear production, the net TKE production stays relatively large. This results in the large dissipation in the WBL at high winds. The LES results show that at hurricane-strength winds more than 90% of the air–sea momentum flux is due to the form
drag of the breakers; that is, the contributions from the surface viscous stress and the nonbreaking wave form drag are small. Our results also suggest that common parameterizations for the mean wind profile and the TKE dissipation used in previous Reynolds-averaged Navier–Stokes WBL models may not be valid.

When the breaker density is high, a large fraction of the WBL is covered with wakes, and the mean wind speed approaches the wind speed inside the wakes. Since breakers are effectively sheltered by other breakers, the wake turbulence is relatively weak. In contrast, when the breaker density is low, the difference between the mean wind and the wind speed inside the wakes becomes large, and the wake turbulence is stronger and becomes significant in the overall WBL turbulence characteristics. Since the sheltering effect can significantly alter the TKE budget, it should be explicitly accounted for in the RANS WBL framework as well.

In open ocean conditions the sea is more developed even at hurricane-strength winds (wave age is typically between 5 and 10, see Moon et al. 2004) and the results of this study are not directly applicable. At larger wave ages the breaking events of the dominant scale waves are likely reduced and the contribution of the form drag from nonbreaking waves becomes increasingly important (Kukulka and Hara 2008b). It is therefore of great interest to investigate to what extent the wake turbulence generation mechanism by breaking waves remains significant in the overall WBL budget over more developed seas. This will be the subject of our next study.

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APPENDIX

Upper and Lower Bounds of Breaking Distribution

We estimate reasonable upper and lower bounds of \( \Lambda(k) \) (integrated in all angles) based on its relationship with the wave saturation spectrum \( B(k) = k^3 \phi(k) \), where \( \phi(k) \) is the wavenumber spectrum (integrated in all angles). If we represent a wave field with a finite number of sinusoidal wave trains of different discrete wavenumbers, then for each wave train \( \phi(k)dk = a^2/k \) where \( a \) is the amplitude of the wave train at \( k \). Thus,

\[
\Delta k/k = \frac{a^2}{2k\phi(k)} = \frac{(ak)^2}{2B(k)}.
\] (A1)

On the other hand, the length of breaking crests per unit horizontal area \( \Lambda(k)\Delta k \) of the same wave train should not exceed the total (breaking and nonbreaking) crest length per unit horizontal area, which equals \( \lambda^2/L \), where \( \lambda \) is the wavelength. Thus, the upper limit of \( \Lambda(k) \) (i.e., when 100% of waves break) may be estimated by

\[
\Lambda(k) < \frac{k}{2\pi \Delta k} = \frac{B(k)}{\pi(ak)^2}. \] (A2)

If we assume that most waves are breaking and the wave slope \( ak \) is close to the critical wave slope 0.3, which is the typical wave slope of breakers, we obtain

\[
\Lambda(k) = \frac{B(k)}{0.09\pi}. \] (A3)

In open ocean conditions under moderate winds, \( B(k) \) is 0.008 ± 0.002 (Romero and Melville 2010) for short gravity waves. In wind wave tanks \( B(k) \) can be as large as 0.1 near the spectral peak (e.g., Caulliez et al. 2008; Jessup and Phadnis 2005). We therefore set the upper and lower bounds of \( B \) as 0.1 and 0.006, and the corresponding upper and lower bounds of \( \Lambda \) as 0.35 and 0.021, as shown in Fig. 3. The figure also shows the laboratory experimental data of microwave breaking with the wind speed of 9.6 m s\(^{-1}\) and peak wavelength of 0.156 m (Jessup and Phadnis 2005). (The uncertainty in their data is due to the uncertainty in the conversion between the measured \( c \) and \( k \) and the uncertainty in the wave age.) Note that the \( \Lambda \) values at wind speeds 40–53 m s\(^{-1}\) (i.e., conditions of this study) are likely higher than the observed values of Jessup and Phadnis (2005) with much lower wind speeds.

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